

Approaches for Achieving Better Comfort and Dehumidification

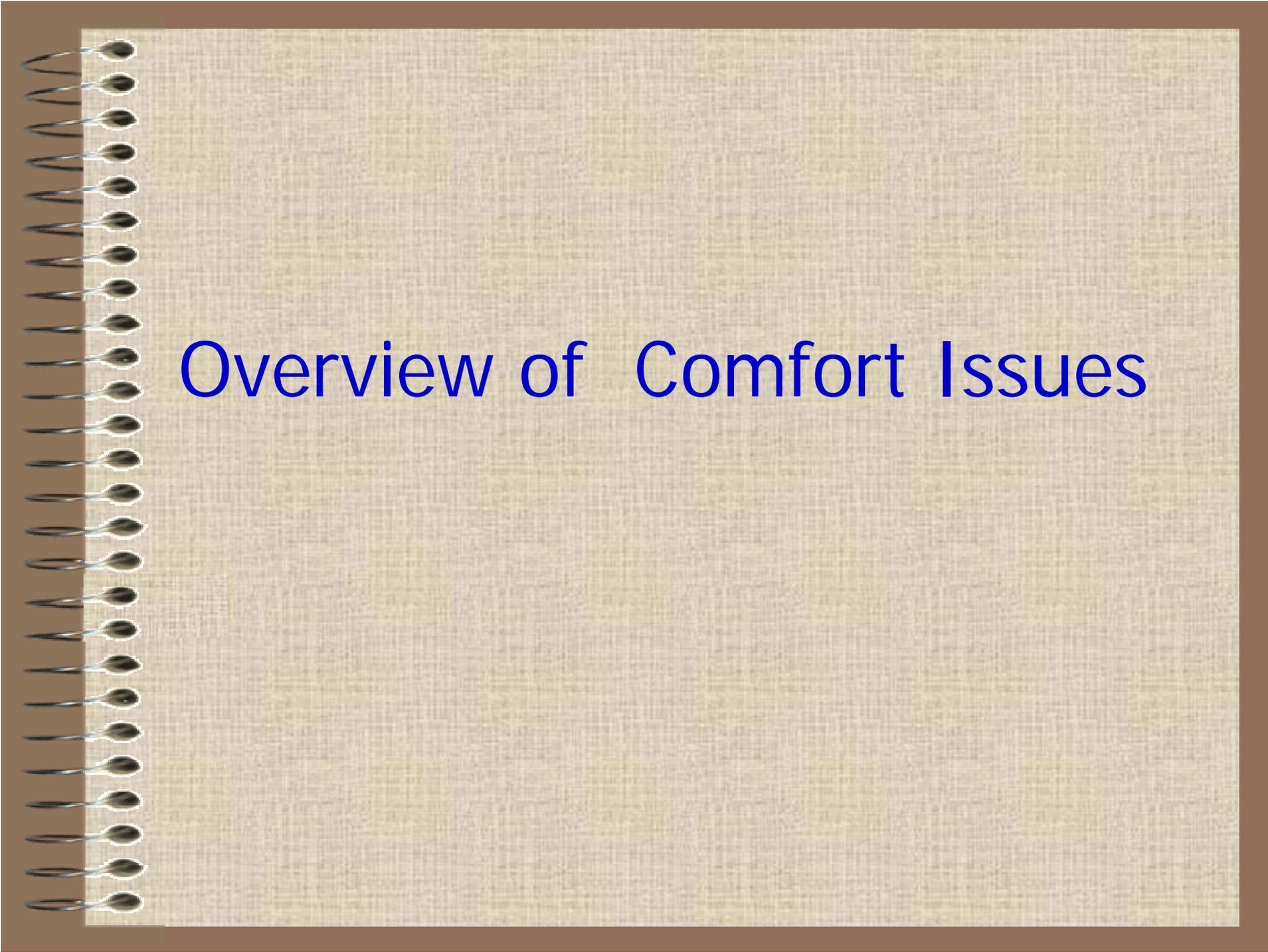
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Presentation Scope

- Overview of Comfort Issues
- Design Process and Considerations to Improve Comfort and Dehumidification Capacity
- Case Studies

A spiral-bound notebook with a brown cover and a light-colored, textured fabric-like surface. The spiral binding is on the left side. The text "Overview of Comfort Issues" is printed in blue on the right side of the cover.

Overview of Comfort Issues

Thermal Comfort

- Condition of Mind that Expresses Satisfaction with Thermal Environment
- Behavioral Actions:
 - Altering Clothing, Activity
 - Change Posture, Location, Thermostat Setting
 - Opening Window, Complain, Leave the Space
- Thermal Comfort Scale
 - + 3 Hot, 0 Neutral, -3 Cold
- ASHRAE Standard 55 - Comfort Zone (Summer, Winter)

Local Discomfort

- Problem: Asymmetric Thermal Radiation, Draft
 - Review: Air Distribution, Location/Quantity/Size of Diffusers
- Problem: Varying Thermal Comfort Scale Through a Building
 - Review: Zoning, Thermostat Location, RA Path Design, Diffuser Selection, Duct Sizing
- Problem: Poor Air Quality, High Humidity
 - Review: Outdoor Air Treatment, AC Sizing/Selection, Type of AC System, Controls, OA/EA Balance, Architectural Issues
- Problem: Noise
 - Review: Diffuser Selection, Duct sizes, Location of AC Units or Terminal Units

Common Problems Resulting in Poor Comfort

- Oversized AC Units (or not Enough Stages for Part-load Comfort Control)
- Poorly Zoned (Low-cost System)
- Improper Location of Thermostat (Behind Refrigerator, Exterior wall, Above Copier etc.,)
- No Outside Air Pre-treatment/Unit Selection with Inadequate Latent Capacity
- Outside Air Closed Due to Inadequate System Capacity
- Air Flow Analysis for Pressure Balances not Performed

Common Problems Resulting in Poor Comfort (Con't)

- Sensors not Routinely Calibrated (Temperature, Humidity, Pressure)
- Poor Selection of Diffusers (NC Levels, Throw, Type)
- Pressure Drop Calculations not Performed Including System Effect
- Renovation Project without Reviewing Air Distribution Issues
- No Reheat Options Provided During Design
- Certified Test and Balance not Performed
- Poor Architectural Design

Common Problems Resulting in Poor Comfort (Con't)

- Flows to AHUs not Balanced
- Sequence of Operations:
 - Never Verified Through Commissioning Process
 - Not Developed for all the Possible Scenarios
 - Poor Communication of Design Intent
- Inadequate/non-existing Preventive Maintenance Programs
- Long-term Reliability/Maintenance Issues

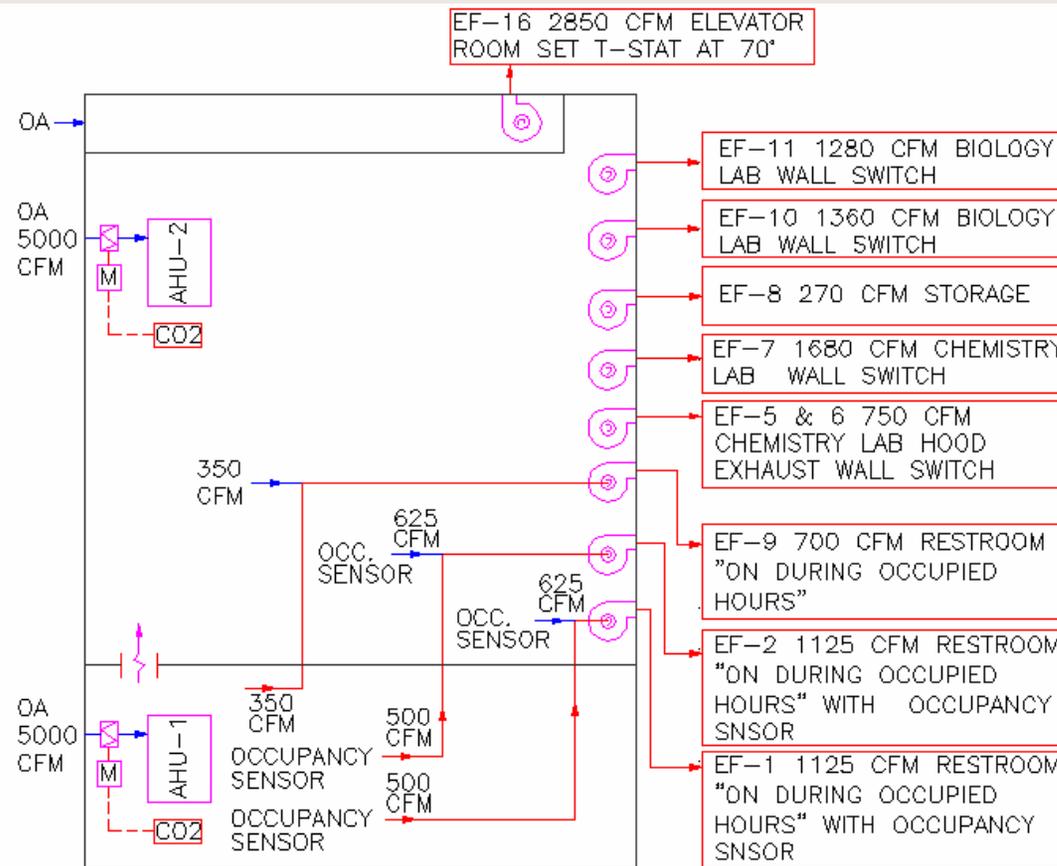
The image shows a spiral-bound notebook with a light beige, textured cover. The spiral binding is on the left side. The title is centered on the cover in a blue, sans-serif font.

Design Process and Considerations to Improve Comfort and Dehumidification

Design Process

- Review and Confirm Requirements
- Perform Zoning Analysis
- Identify Ventilation Rates Per Code
- Air Balance Diagrams (Outside Air/Exhaust Air)
- Minimize Impacts with:
 - CO₂ Control
 - Motorized Dampers
 - Occupancy Sensors
- Review Sensible Heat Ratio
 - Select Appropriate System Based on SHR

Typical Air Balance Diagram



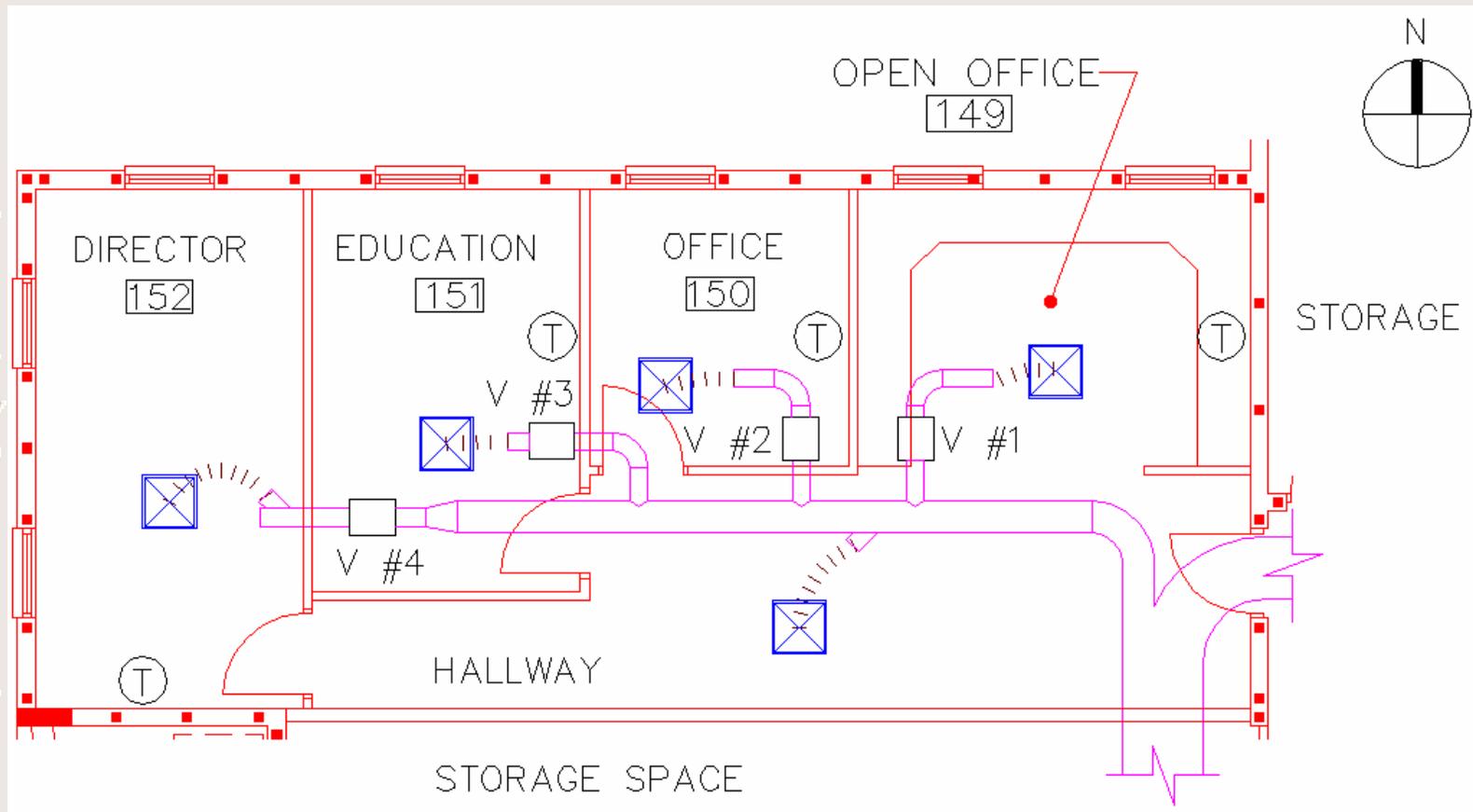
Design Process (Con't)

- Review Applicable System Options
 - Constant Volume/Variable Air Volume
 - Single Zone/Multi-zone
 - DX /Chilled Water
 - Additional Enhancements
- Select System Based on
 - Capital Cost (Budget)
 - Comfort/Process Requirements
 - Space Restrictions, Maintenance, Energy Use
- Air Distribution System and Controls Design

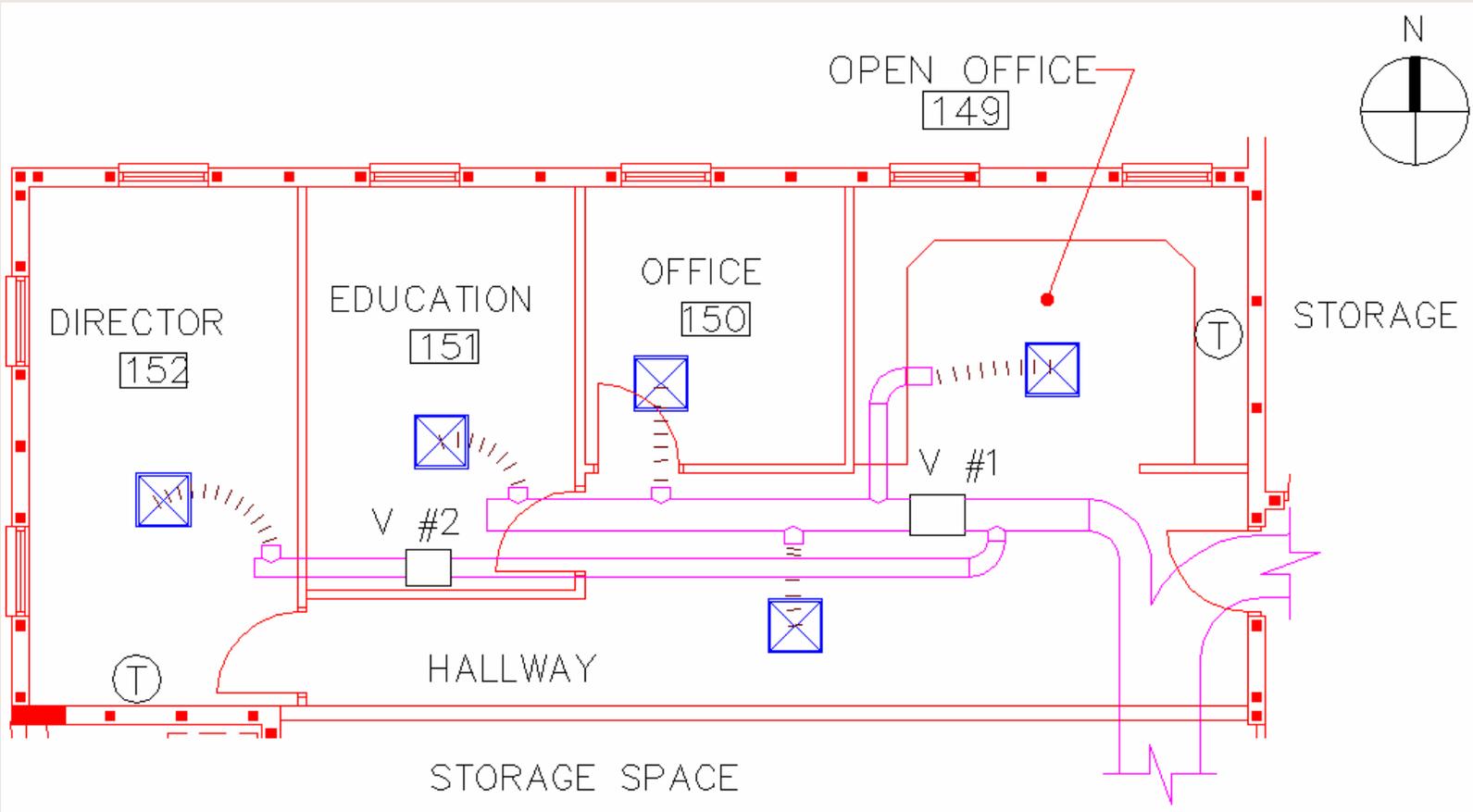
Zoning

- Perimeter and Core Zones
 - Fan Powered Boxes/VAV
- Occupancy (Meeting Rooms, Class Rooms)
- Process Requirements (Clean Area, Computer Rooms, Chemical Labs)
- Temperature/Humidity Control Requirements
- Sensors Location
 - Supply
 - Return
 - Space

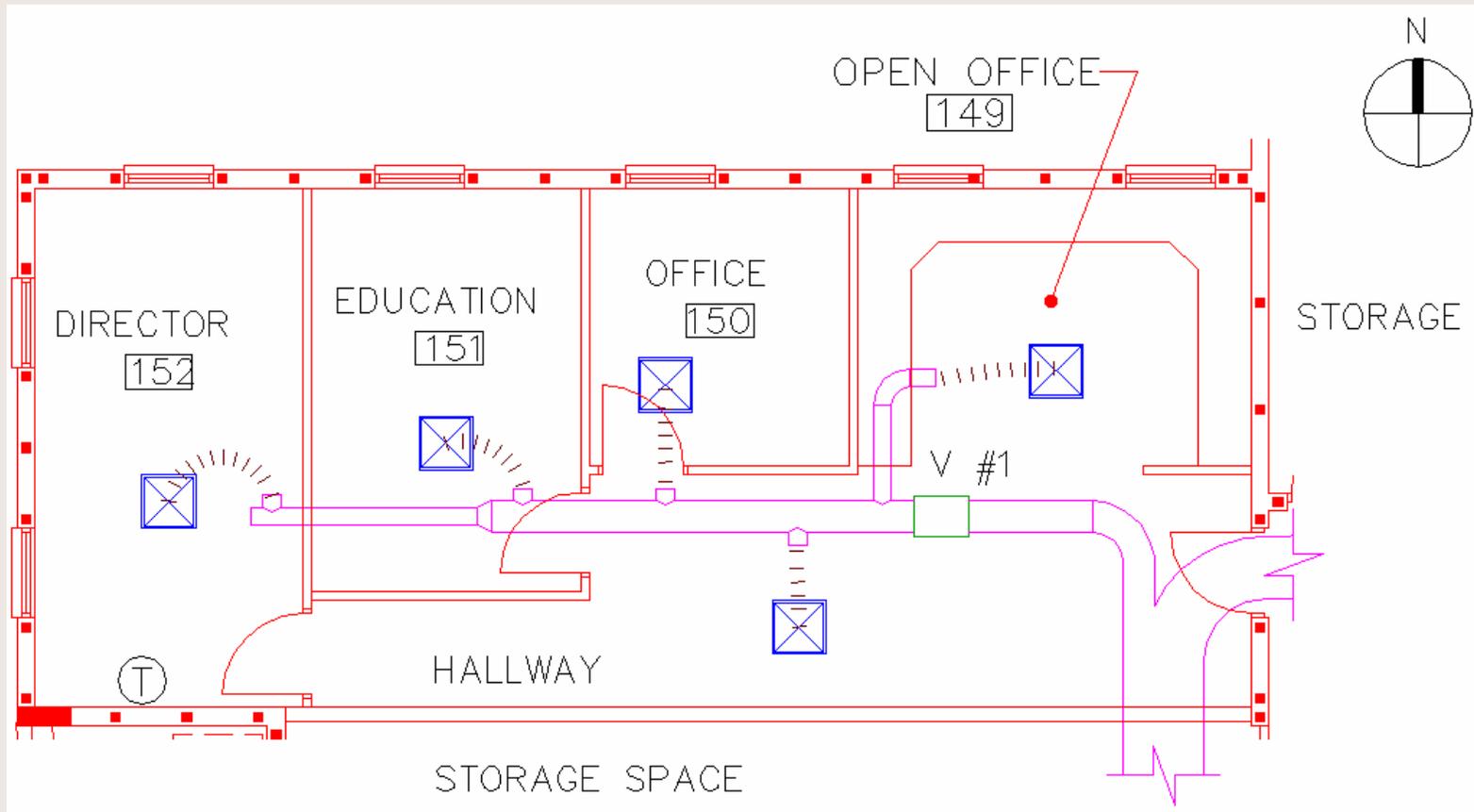
Thermostat Locations - VAVs



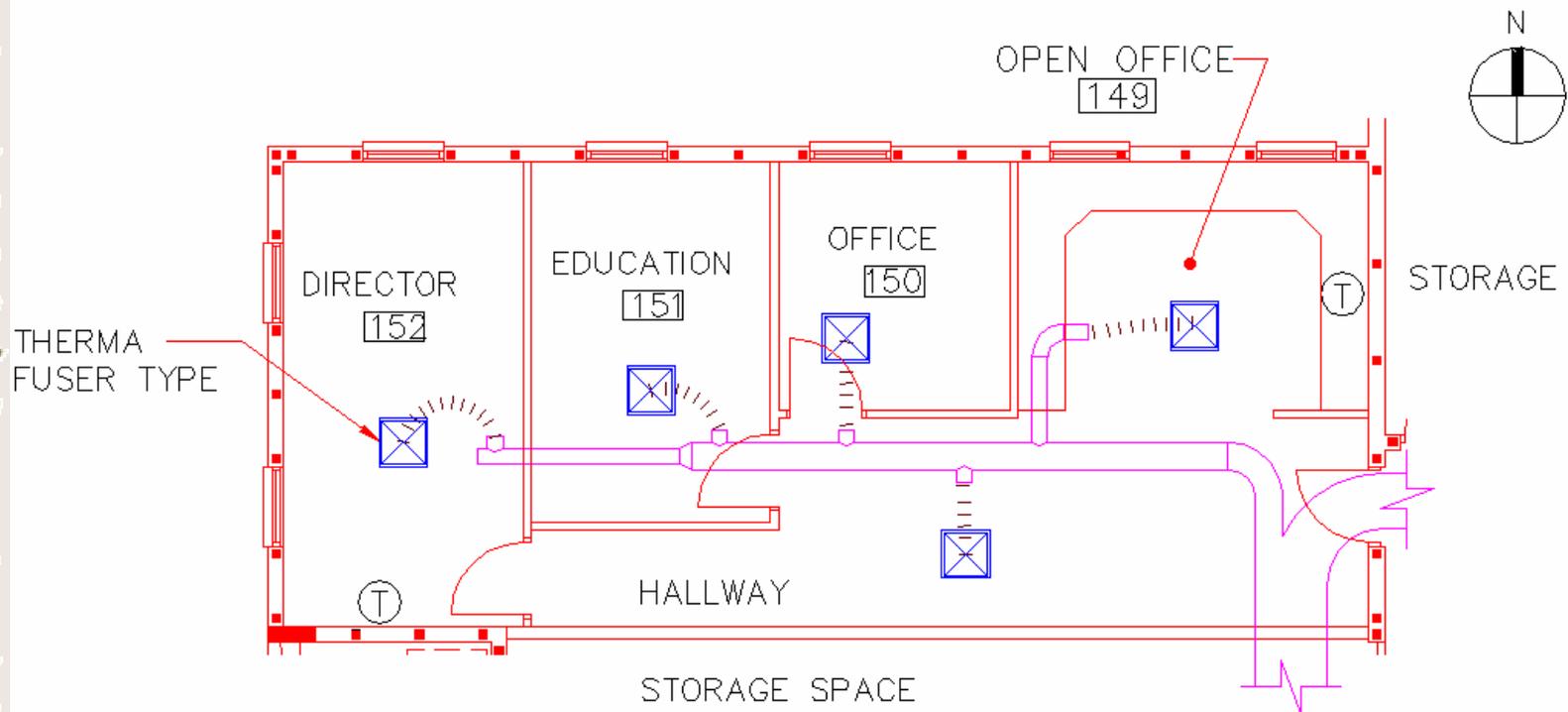
VAVs - Two Zones



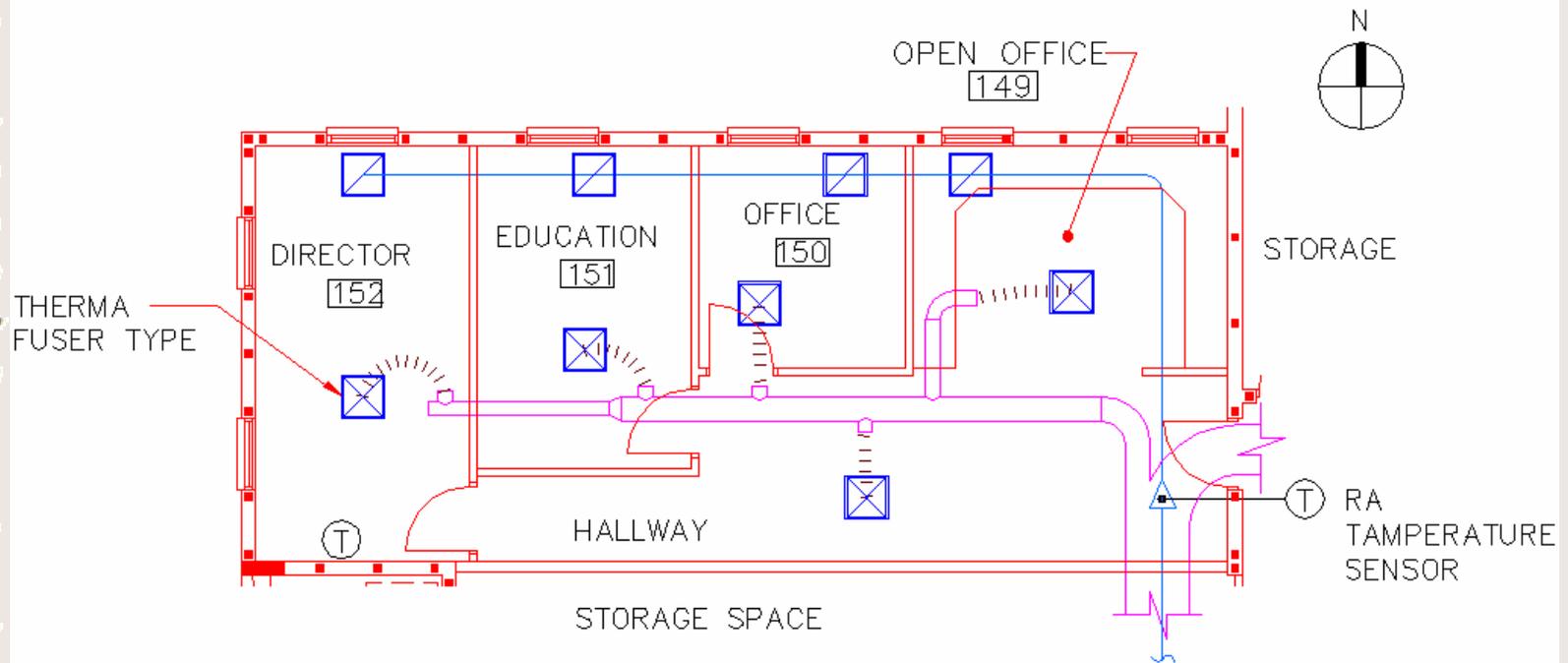
VAV - One Zone



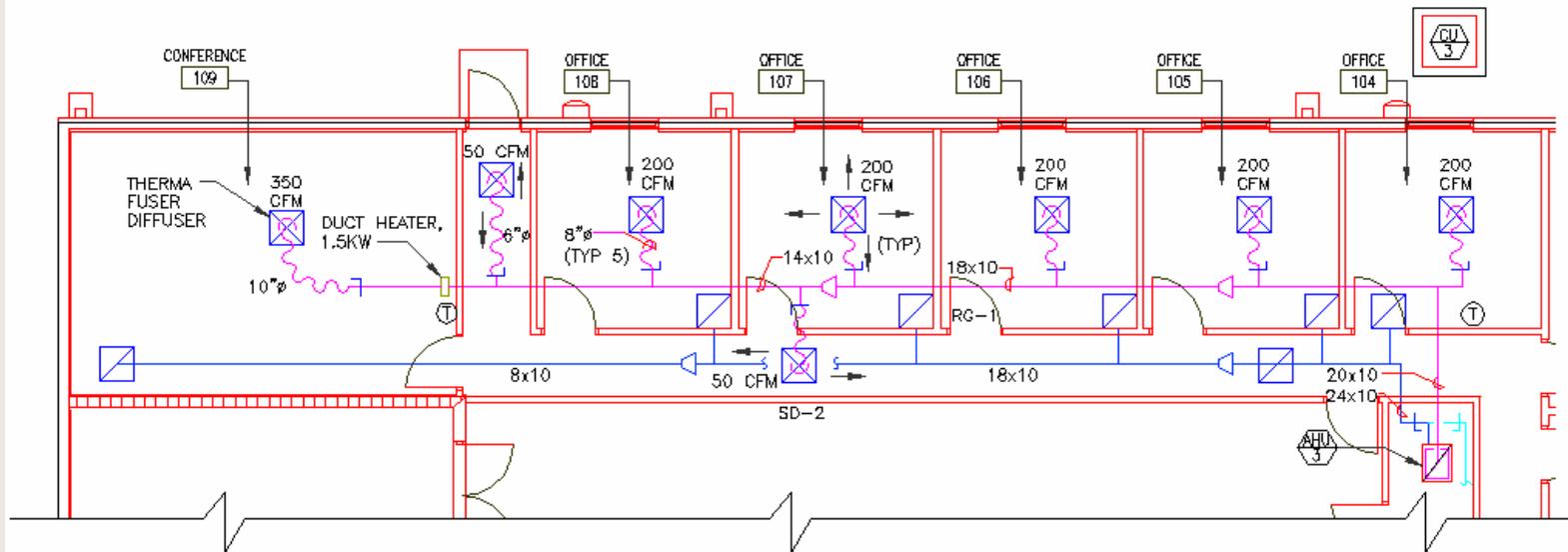
Thermostats – AHU



Thermostats – RA & Space



Typical Office – HVAC Control



TYPICAL OFFICE LAYOUT
N.T.S.

Air Distribution

- Separate OA and SA Distribution
- Mix Conditioned OA and SA Upstream/Downstream
- VAV - Fan Powered Boxes to Improve Circulation
- Higher Air Flow for Perimeter Diffusers in a Single Space
- High Induction Diffuser for Low air Flow Spaces
- Spacing of Diffusers and ADPI Calculations
- Ensure Return Air Flow Path from Each Space
- Ducted Versus Plenum Return
- Constant-Fan Operation/Auto-Fan Operation

SA/OA Requirements Analysis

- OA Requirement per ASHRAE 62-2001 - 2000 CFM
- If Critical Fraction to Space 3 Reduced to 0.375, OA - 1030 CFM
- Increases Reheat, Minimizes Capital Cost
- Reduces Dehumidification Costs in Humid Climates

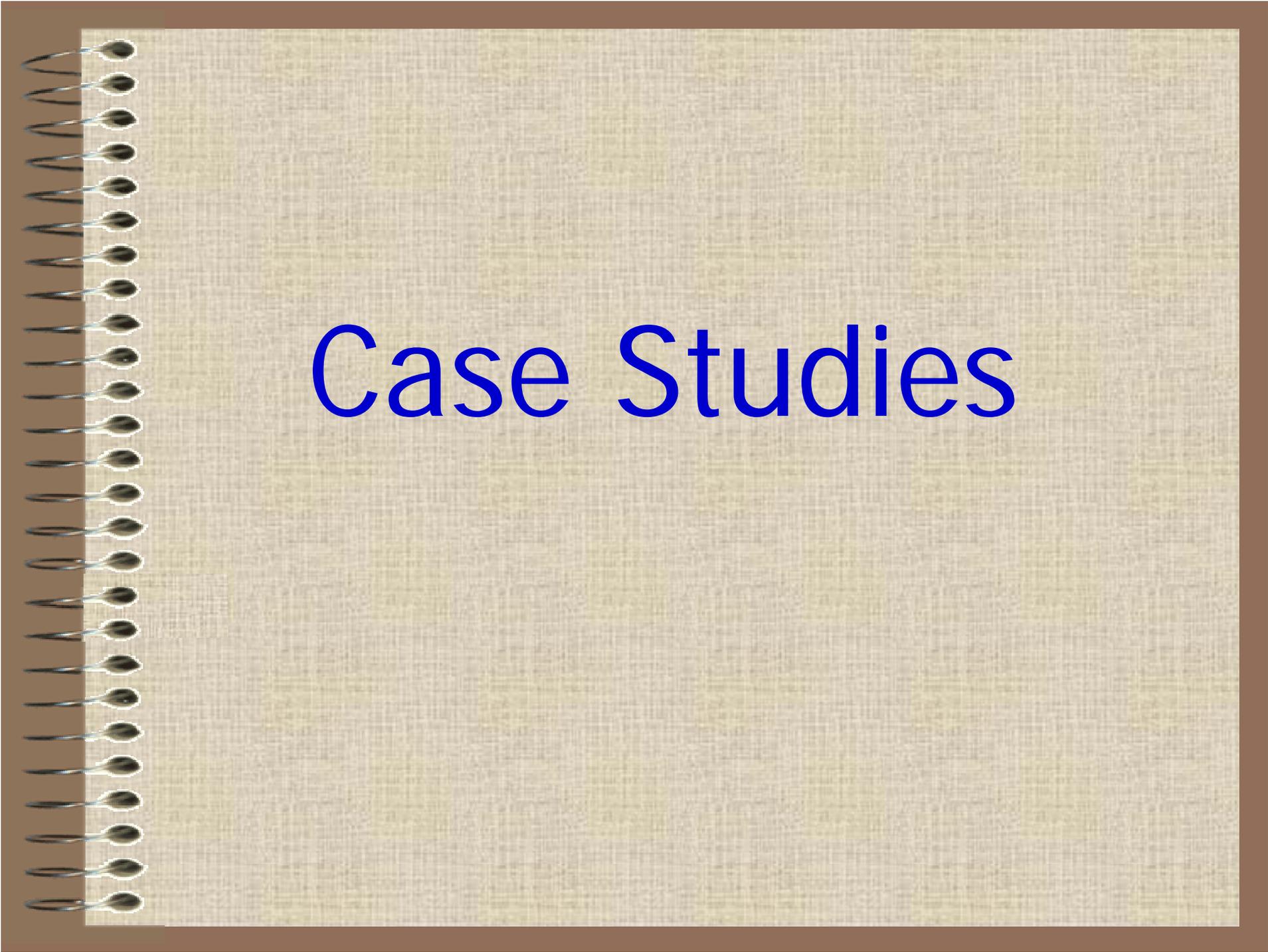
Description	Space 1	Space 2	Space 3
Supply Air CFM	800	800	400
Outside Air CFM	200	300	400
Fraction of OA	0.25	0.375	1.0

Dehumidification

- Active Versus Passive Humidity Control
- Provide Source Control/Treatment for Latent Loads
 - Demand Controlled Ventilation
 - Positive Pressure in Spaces (Reduce Infiltration)
 - Pre-treat Outside Air
 - Utilize Energy Recovery Wheels, Heat Pipes, Desiccant Systems
- Utilize the State-of-the Art Controls Capability
- Utilize Multiple Condensing Units or Multiple Compressors
- Face and Bypass Option, Face-split coils
- Avoid Utilizing Fan Coil Systems in Humid Climates
- Use Lower Supply CFM (275 to 350 CFM per Ton)
- Lower Supply Air Temperature (48 to 52 deg F) and Control SAT

Where/Why VAV More Suited?

- Modulating loads
- Better Zoning Capabilities
- Better Humidity Control with Constant SAT
- Lower Operating Costs
- More Forgiving if Unit is Oversized
- DDC Controls Help to Control VAV & Meet Ventilation

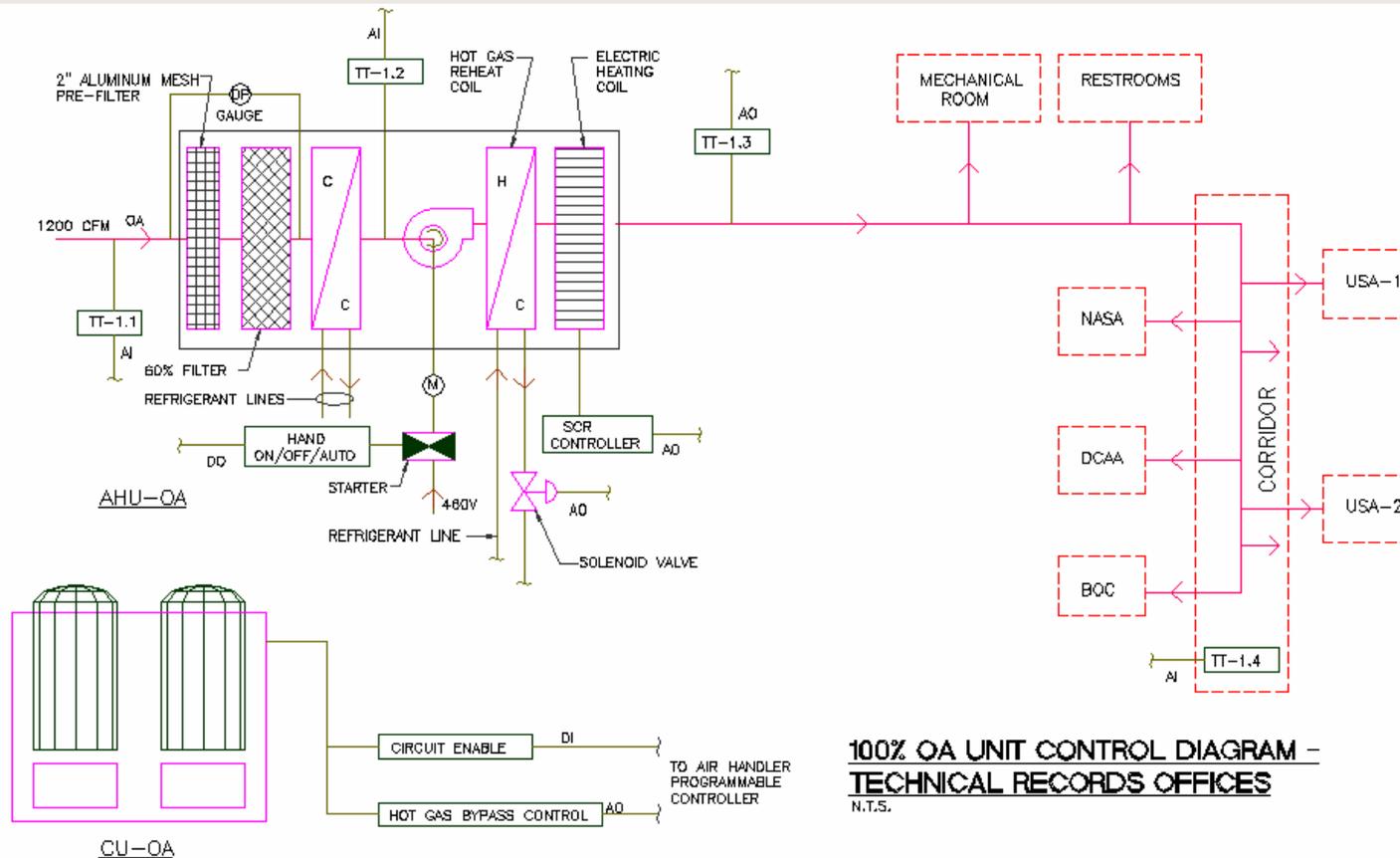
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Case Studies

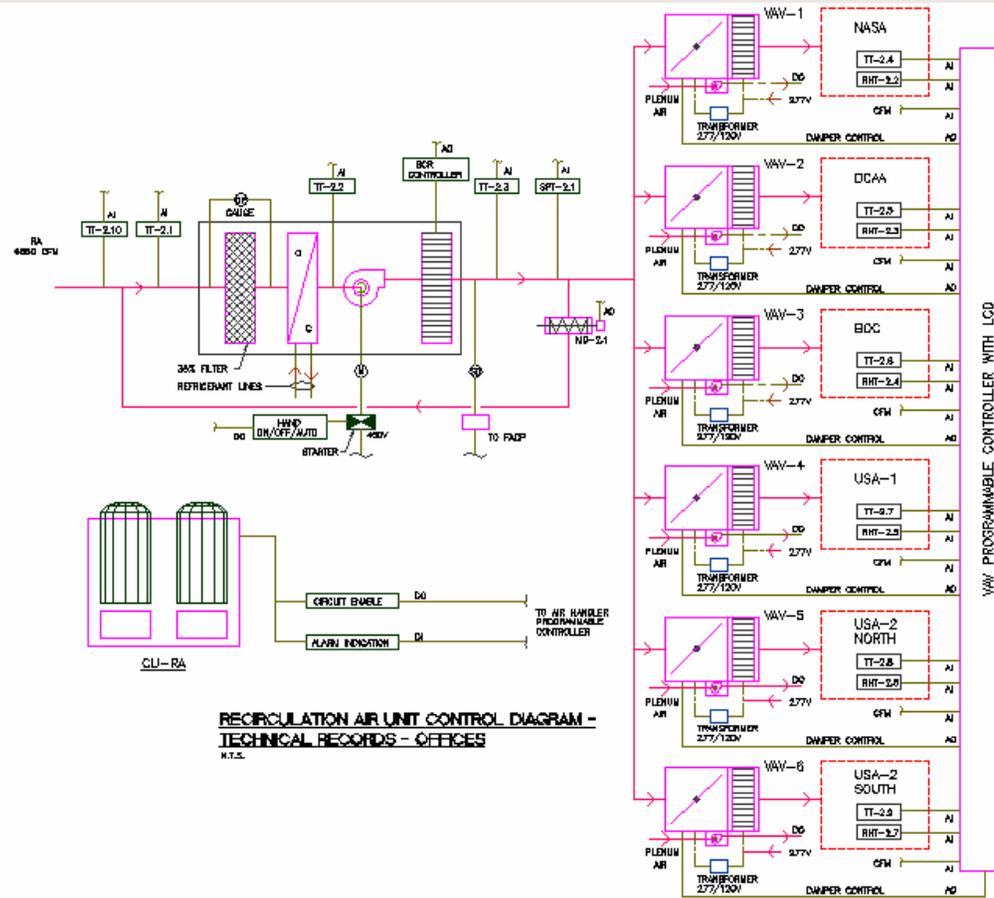
Case Study 1

- Technical Records Facility (14,000 Sq.ft.)
- Multiple Users
- Pre-Engineered Metal Building
- Dual-path System
 - CAV OA Unit (7.5 Tons)
 - VAV DX unit (20 Tons)
- Active Temperature and Humidity Control
- Positive Pressurization

Dedicated OA Unit



Recirculation Unit



Design Highlights

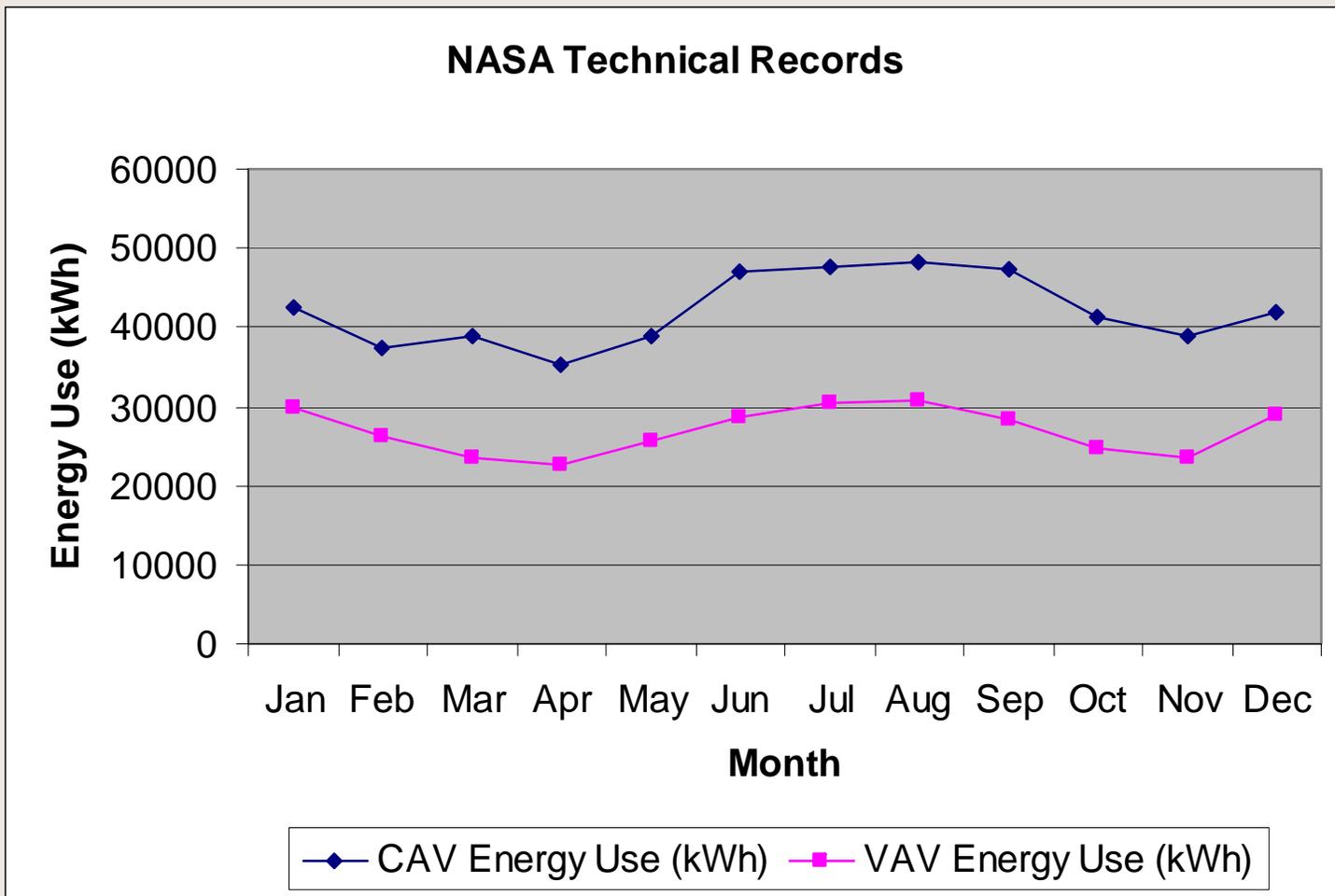
- Supply Air Fan Cycling (Reduced Moisture Carryover)
- Fan Powered VAV - Humidity Control Capability
- Electric Heat/Hot Gas-Reheat
- Hot Gas Bypass - 53 +/- 3 deg F Control
- 35% Energy Savings Compared to a Constant Volume System
- Humidity Control - 50 +/- 5% RH for the Last Three Years
- Excellent Comfort Control (ASHRAE Regional Technology Award)

NASA Technical Records

HVAC



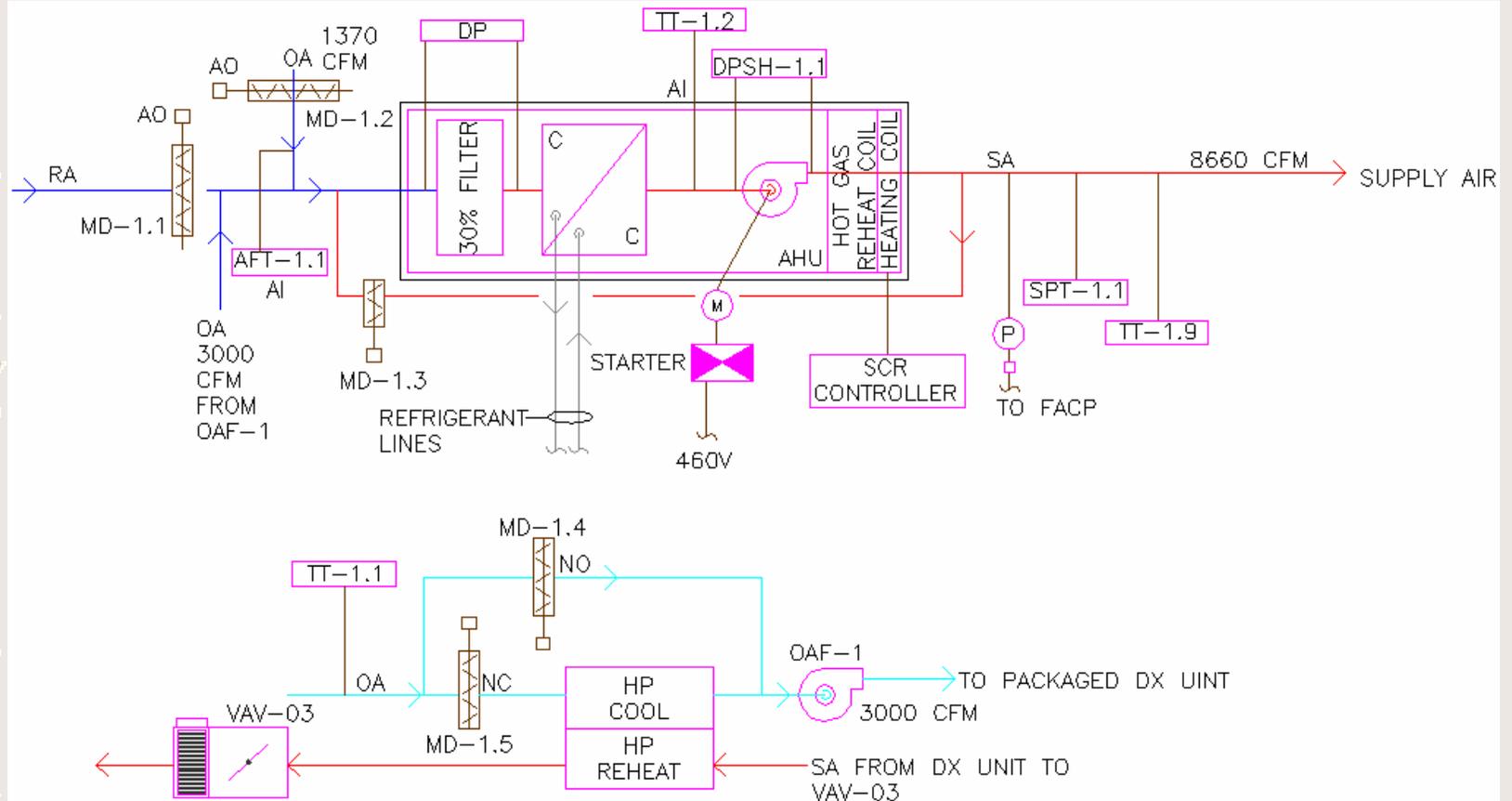
Energy Use Analysis



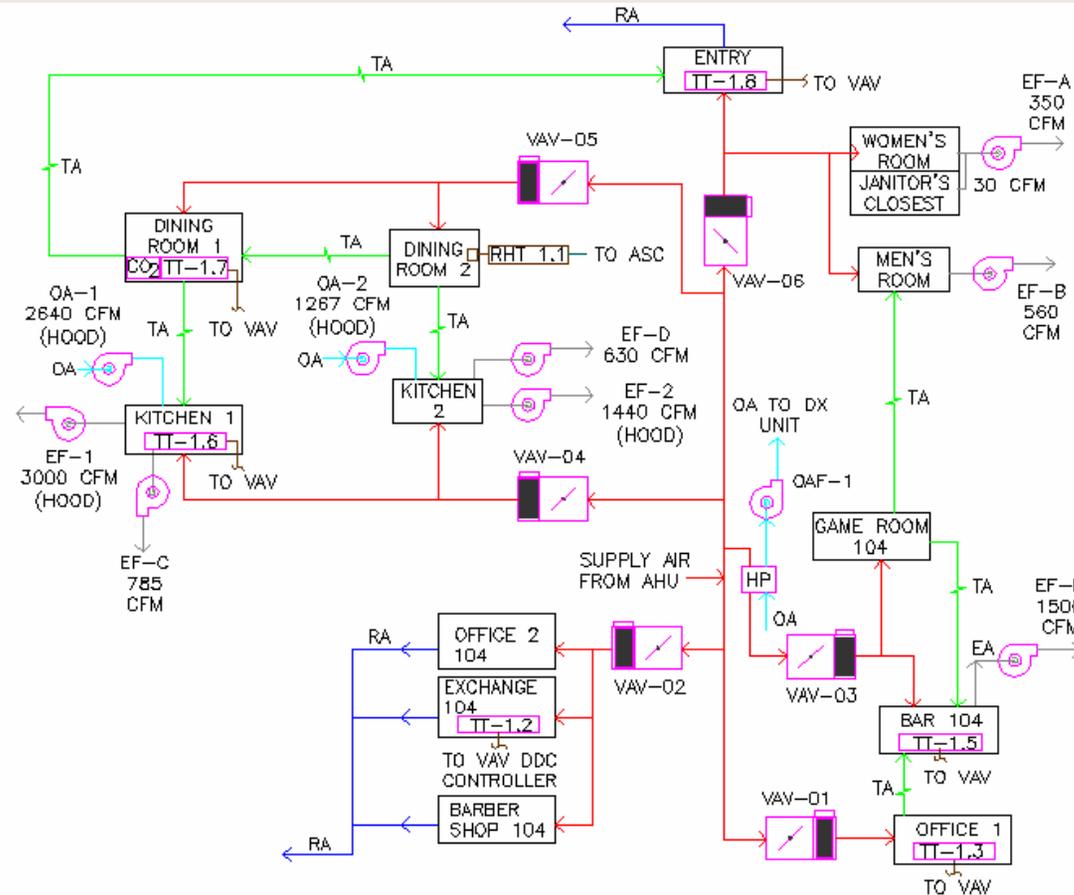
Case Study 2

- Multi-user Facility 3600 sq.ft
- Dining, Kitchen, Game Room, Retail Space, Office and Barber shop
- VAV System - Bypass Damper
- Pressure Balance, Outside Air control, Heat Recovery
- DDC - Temperature and Humidity Controls
- 25 Tons Split-AC Located Outside the Facility

Multi-user Facility HVAC



Multi-user Facility Comfort Control



Design Highlights

- Constant Supply air Temperature to Improve Dehumidification Capacity at all Times
- Heat Pipes to Reheat a Zone with High OA Requirement and Pre-cool Outside Air
- CO₂ Sensor to Modulate Ventilation in Dining Space
- Active Humidity Control through a Space RH Sensor in Dining Space
- OA/EA Flow Balance to Various Zones Under Various Operating Conditions

Design Highlights

- Minimize Energy Use through Hot gas Reheat Control When Required
- DDC Controls with Sequences Identified for Various Operating Scenarios
- Has Been in Operation for the Last Three years Providing Excellent Comfort Conditions

Old HVAC – Multi-user Facility



Main HVAC Unit



North Dining Room Condensing Unit



Pool Room Unit



2 ton Wall-Hung Unit

New HVAC Units



Summary

- Energy Efficiency and Improved Comfort Control can be Achieved Through an Integrated Design Process
- Current DDC Controls Provide Several Options to Better Control Individual Zones
- Cost-effective Technologies are Available to Improve Dehumidification Capacity of Units
- Identifying/Understanding Requirements is the Most Critical Step in Designing a Cost-effective HVAC System

Thank You!

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