

Approaches for Achieving Better Comfort and Dehumidification

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Presentation Scope

- Overview of Comfort Issues
- Design Process and Considerations to Improve Comfort and Dehumidification Capacity
- Case Studies

Overview of Comfort Issues

Thermal Comfort

- Condition of Mind that Expresses Satisfaction with Thermal Environment
- Behavioral Actions:
 - Altering Clothing, Activity
 - Change Posture, Location, Thermostat Setting
 - Opening Window, Complain, Leave the Space
- Thermal Comfort Scale
+ 3 Hot, 0 Neutral, -3 Cold
- ASHRAE Standard 55 - Comfort Zone (Summer, Winter)

Local Discomfort

- Problem: Asymmetric Thermal Radiation, Draft
 - Review: Air Distribution, Location/Quantity/Size of Diffusers
- Problem: Varying Thermal Comfort Scale Through a Building
 - Review: Zoning, Thermostat Location, RA Path Design, Diffuser Selection, Duct Sizing
- Problem: Poor Air Quality, High Humidity
 - Review: Outdoor Air Treatment, AC Sizing/Selection, Type of AC System, Controls, OA/EA Balance, Architectural Issues
- Problem: Noise
 - Review: Diffuser Selection, Duct sizes, Location of AC Units or Terminal Units

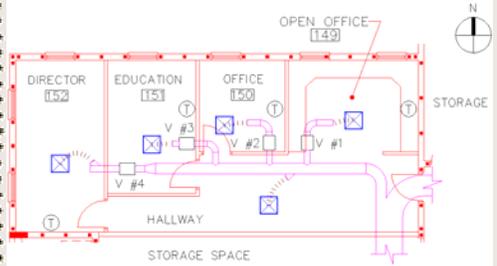
Common Problems Resulting in Poor Comfort

- Oversized AC Units (or not Enough Stages for Part-load Comfort Control)
- Poorly Zoned (Low-cost System)
- Improper Location of Thermostat (Behind Refrigerator, Exterior wall, Above Copier etc..)
- No Outside Air Pre-treatment/Unit Selection with Inadequate Latent Capacity
- Outside Air Closed Due to Inadequate System Capacity
- Air Flow Analysis for Pressure Balances not Performed

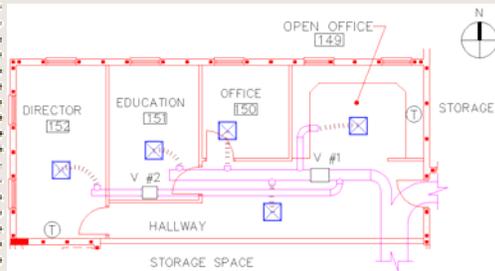
Zoning

- Perimeter and Core Zones
 - Fan Powered Boxes/VAV
- Occupancy (Meeting Rooms, Class Rooms)
- Process Requirements (Clean Area, Computer Rooms, Chemical Labs)
- Temperature/Humidity Control Requirements
- Sensors Location
 - Supply
 - Return
 - Space

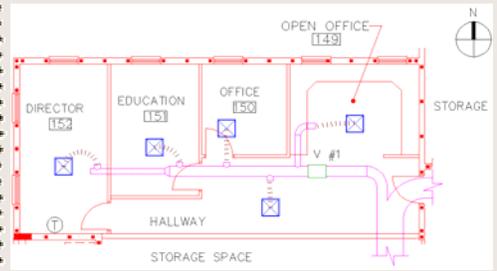
Thermostat Locations - VAVs



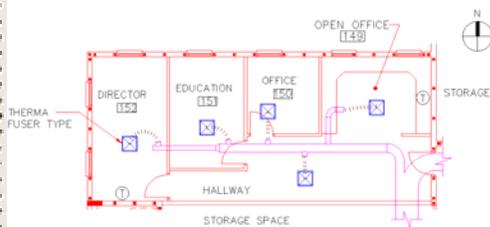
VAVs - Two Zones



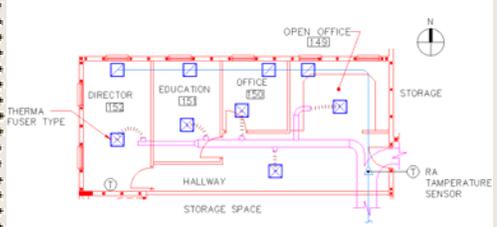
VAV - One Zone



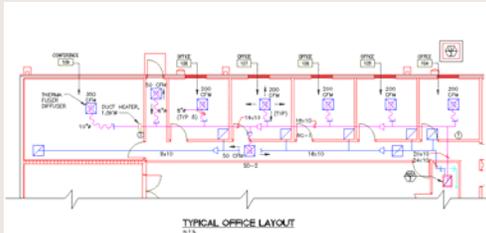
Thermostats – AHU



Thermostats – RA & Space



Typical Office – HVAC Control



Air Distribution

- Separate OA and SA Distribution
- Mix Conditioned OA and SA Upstream/Downstream
- VAV - Fan Powered Boxes to Improve Circulation
- Higher Air Flow for Perimeter Diffusers in a Single Space
- High Induction Diffuser for Low air Flow Spaces
- Spacing of Diffusers and ADPI Calculations
- Ensure Return Air Flow Path from Each Space
- Ducted Versus Plenum Return
- Constant-Fan Operation/Auto-Fan Operation

SA/OA Requirements Analysis

- OA Requirement per ASHRAE 62-2001 - 2000 CFM
- If Critical Fraction to Space 3 Reduced to 0.375, OA - 1030 CFM
- Increases Reheat, Minimizes Capital Cost
- Reduces Dehumidification Costs in Humid Climates

Description	Space 1	Space 2	Space 3
Supply Air CFM	800	800	400
Outside Air CFM	200	300	400
Fraction of OA	0.25	0.375	1.0

Dehumidification

- Active Versus Passive Humidity Control
- Provide Source Control/Treatment for Latent Loads
 - Demand Controlled Ventilation
 - Positive Pressure in Spaces (Reduce Infiltration)
 - Pre-treat Outside Air
 - Utilize Energy Recovery Wheels, Heat Pipes, Desiccant Systems
- Utilize the State-of-the Art Controls Capability
- Utilize Multiple Condensing Units or Multiple Compressors
- Face and Bypass Option, Face-split coils
- Avoid Utilizing Fan Coil Systems in Humid Climates
- Use Lower Supply CFM (275 to 350 CFM per Ton)
- Lower Supply Air Temperature (48 to 52 deg F) and Control SAT

Where/Why VAV More Suited?

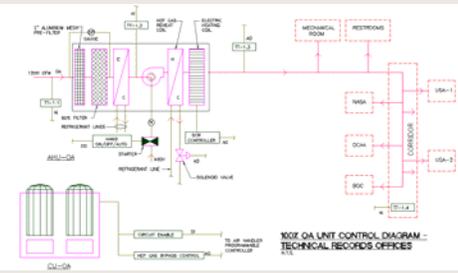
- Modulating loads
- Better Zoning Capabilities
- Better Humidity Control with Constant SAT
- Lower Operating Costs
- More Forgiving if Unit is Oversized
- DDC Controls Help to Control VAV & Meet Ventilation

Case Studies

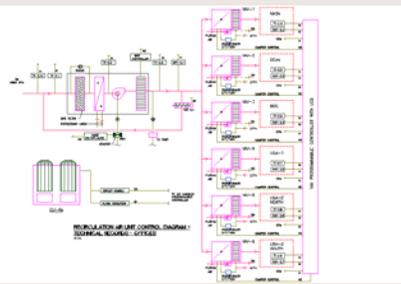
Case Study 1

- Technical Records Facility (14,000 Sq.ft.)
- Multiple Users
- Pre-Engineered Metal Building
- Dual-path System
 - CAV OA Unit (7.5 Tons)
 - VAV DX unit (20 Tons)
- Active Temperature and Humidity Control
- Positive Pressurization

Dedicated OA Unit



Recirculation Unit



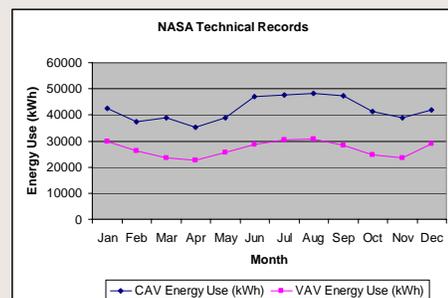
Design Highlights

- Supply Air Fan Cycling (Reduced Moisture Carryover)
- Fan Powered VAV - Humidity Control Capability
- Electric Heat/Hot Gas-Reheat
- Hot Gas Bypass - 53 +/- 3 deg F Control
- 35% Energy Savings Compared to a Constant Volume System
- Humidity Control - 50 +/- 5% RH for the Last Three Years
- Excellent Comfort Control (ASHRAE Regional Technology Award)

NASA Technical Records HVAC



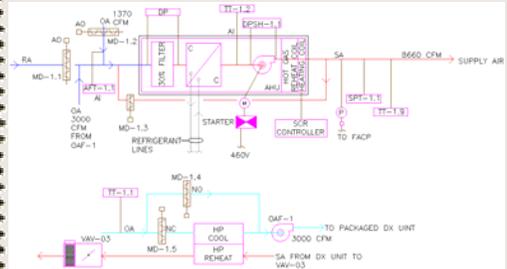
Energy Use Analysis



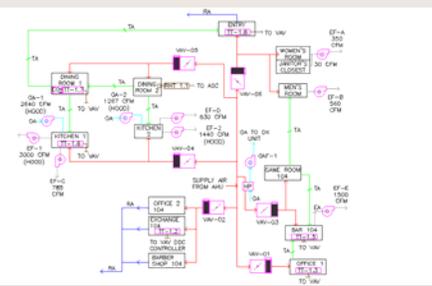
Case Study 2

- Multi-user Facility 3600 sq.ft
- Dining, Kitchen, Game Room, Retail Space, Office and Barber shop
- VAV System - Bypass Damper
- Pressure Balance, Outside Air control, Heat Recovery
- DDC - Temperature and Humidity Controls
- 25 Tons Split-AC Located Outside the Facility

Multi-user Facility HVAC



Multi-user Facility Comfort Control



Design Highlights

- Constant Supply air Temperature to Improve Dehumidification Capacity at all Times
- Heat Pipes to Reheat a Zone with High OA Requirement and Pre-cool Outside Air
- CO₂ Sensor to Modulate Ventilation in Dining Space
- Active Humidity Control through a Space RH Sensor in Dining Space
- OA/EA Flow Balance to Various Zones Under Various Operating Conditions

Design Highlights

- Minimize Energy Use through Hot gas Reheat Control When Required
- DDC Controls with Sequences Identified for Various Operating Scenarios
- Has Been in Operation for the Last Three years Providing Excellent Comfort Conditions

Old HVAC – Multi-user Facility



New HVAC Units



Summary

- Energy Efficiency and Improved Comfort Control can be Achieved Through an Integrated Design Process
- Current DDC Controls Provide Several Options to Better Control Individual Zones
- Cost-effective Technologies are Available to Improve Dehumidification Capacity of Units
- Identifying/Understanding Requirements is the Most Critical Step in Designing a Cost-effective HVAC System

Thank You!

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